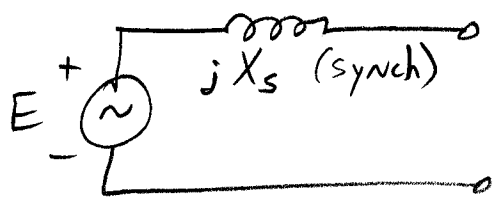
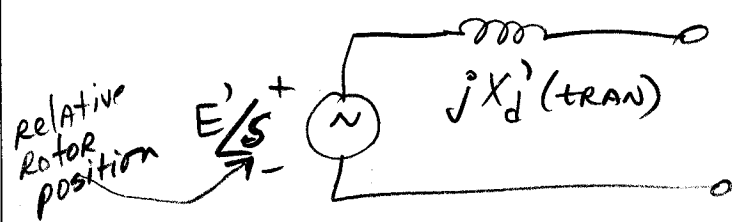


Generator Representation

Accurately modeled using transformation of variables, into a reference frame rotating at synch speed. Internal impedances are actually in operational form $X(s)$. Time constants involved. Sometimes simplified as



Steady State



TRANSIENT
5 to 30 cycles

Note constant E, E' . Constant voltage behind synch reactance behind TRANS reactance

Time constants for $E' \sim 1/2$ second

Assume voltage regulator, governor, boiler controls unchanged.

Swing Equation

SIA on back

Rotating Mass with no damping

$$J \alpha = T_A = T_m - T_e \quad (1)$$

J: Moment of inertia - kg-m² α : Angular acceleration - rad/s²T_A: Net accelerating torque - newt-mT_m: Applied mechanical torque (from external sys)T_e: Produced electrical torque (to external sys)

$$(\text{kg m}^2) \frac{\text{rad}}{\text{s}^2} = \text{N-m}$$

$$\text{kg} \frac{\text{m}^2}{\text{s}^2} \rightarrow \left(\frac{\text{kg m}}{\text{s}^2} \right) \text{m}$$

← units of Newton

Define $P = T\omega$ $(\text{N-m}) \left(\frac{\text{R}}{\text{s}} \right) \rightarrow \frac{\text{N-m}}{\text{s}}$ rate of doing work

$$\omega = \dot{\theta}, \quad \alpha = \dot{\omega} = \ddot{\theta}$$

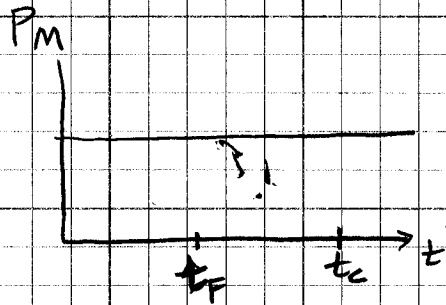
$$\frac{d\theta}{dt}, \quad \frac{d\omega}{dt}, \quad \frac{d^2\theta}{dt^2}$$

$$J \frac{d^2\theta}{dt^2} = T_m - T_e, \quad (J\omega) \frac{d^2\theta}{dt^2} = P_m - P_e$$

↑
Mech
input
power↑
Generated
Elect
Power

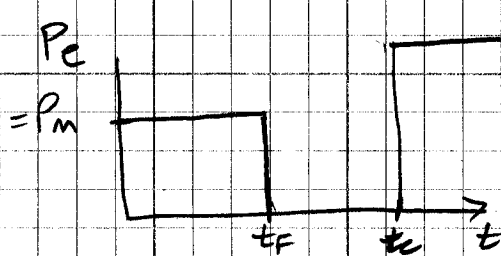
Define $H = \frac{\text{Stored kinetic energy at synch speed}}{\text{machine rating in MVA}}$

$$= \frac{\frac{1}{2} J \omega_s^2}{S_{MB}} = (J \omega_s) \frac{\omega_s}{2 S_{MB}} = H$$



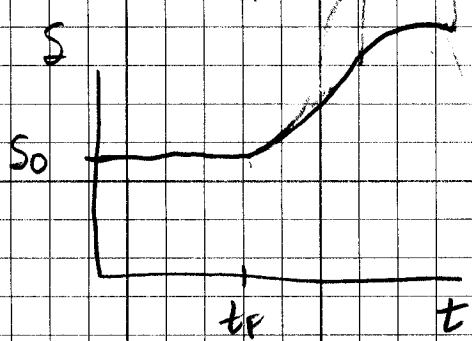
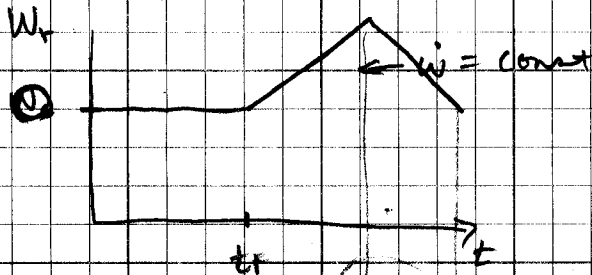
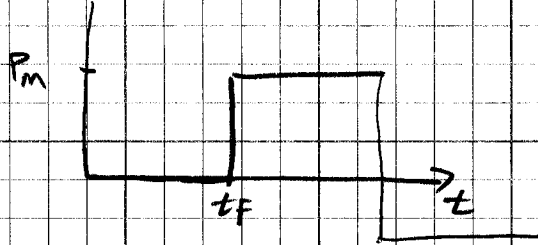
$$\frac{d^2s}{dt^2} = \frac{P_A W_s}{2H}$$

$$\ddot{s} = \dot{w} = \frac{P_A W_s}{2H}$$



New-higher level
due to S
(pretend const)

$$P_A = P_m - P_e$$



AAZ/2

Actually, $W \neq W_s$, but % variation very small.

Let $JW = JW_s \rightarrow JW = \frac{ZHS_{MB}}{W_s}$

Now, $\frac{ZHS_{MB}}{W_s} \frac{d^2\theta}{dt^2} = P_m - P_e$

$\frac{ZH}{W_s} \frac{d^2\theta}{dt^2} = \frac{P_m}{S_{MB}} - \frac{P_e}{S_{MB}} = P_m - P_e$] per unit

generated power
 325MW
 93ft long, 178 tons,
 on 7 bearings,
 3RPM turning gear for
 72 hours before stop

H for turbines (on own base) 4-9

Takes a long time to roll to a stop ~ 45min

Spins in Hydrogen

↳ 70% of air density

Better cooling - Would burn up in air - red hot tips
 13ft diam wheel @ 3600

Pressurized system - no moisture, oxygen

Not explosive if > 70% hydrogen
 check > 94% H

$W = 60(2\pi) = 377 \text{ r/s}$
 $R = 6\frac{1}{2}' = 1.98 \text{ m}$
 $WR = 746.5 \text{ m/s}$
 $= 1700 \text{ MPH}$
 $> \text{MACH } 2$

Of course, θ is always increasing

$\theta(t) = \int W dt + \theta_0$
 ↑
 synch

Steady state $\rightarrow P_m = P_e, \frac{d^2\theta}{dt^2} = 0$

Important quantity is movement of θ from synch θ .

Let $\theta(t) = \underbrace{W_s t + \theta_0}_{\text{steady state}} + S(t)$

$\frac{d\theta(t)}{dt} = W_s + \frac{dS(t)}{dt}$, $\frac{d^2\theta(t)}{dt^2} = \frac{d^2S(t)}{dt^2}$

$$P_{Mech} = P_{Elec} + \frac{d}{dt} W_{kinetic} + P_{FRICITION}^{SMALL} \quad (1)$$

$$W_{kinetic} = \frac{1}{2} J \omega^2 = \frac{1}{2} J (\dot{\theta})^2$$

$$\frac{dW_{kinetic}}{dt} = J \omega \dot{\omega} = J \omega \ddot{\delta} \approx J \omega_0 \ddot{\delta}$$

$$So \quad P_{mech} = P_{Elec} + J \omega_0 \ddot{\delta}$$

$$\ddot{\delta} = \frac{P_{mech} - P_{Elec}}{J \omega_0}$$

$$\frac{P_{mech}}{S_{B3\phi}} = \frac{P_{Elec}}{S_{B3\phi}} + \frac{J \omega_0 \ddot{\delta}}{S_{B3\phi}}$$

$P_{M,pu} \quad P_{E,pu}$

$$\frac{2(\frac{1}{2} J \omega_0^2)}{\omega_0 S_{B3\phi}}$$

$$= \frac{2 W_{kinetic}}{\omega_0 S_{B3\phi}} \leftarrow H \text{ sec.}$$

$$= \frac{2H}{\omega_0} = \frac{H}{\omega_0/2}$$

$$= \frac{H}{2\pi f_0/2} = \frac{H}{\pi f_0} = M$$

$$So \quad P_{mech, pu} - P_{Elec, pu} = M \ddot{\delta}$$

$$\ddot{\delta} = \frac{P_{mech, pu} - P_{Elec, pu}}{M}$$

The swing Equation

$$\dot{\omega} = \dot{\omega}_{REL}$$

$$\omega_{REL} = \omega - \omega_0$$

Before a disturbance, ω_{REL} is zero.

After a disturbance, ω_{REL} becomes zero again if stable (see curve)

$$\int_0^{t_{smax}} \omega_{REL} dt = 0$$

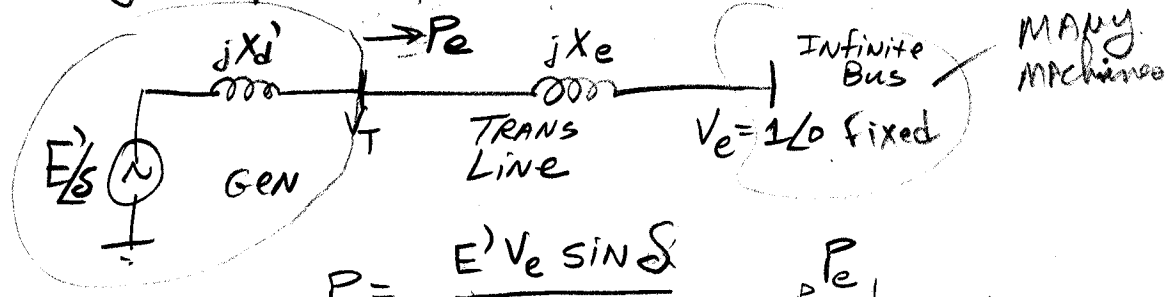
Then

$$\frac{2H}{\omega_s} \frac{d^2\delta}{dt^2} = P_m - P_e$$

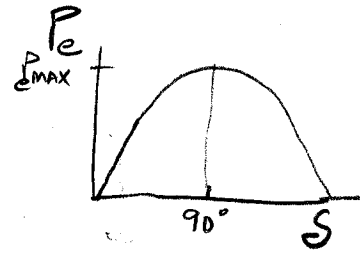
Per Unit swing eq.
(No damping)

δ is departure from synch reference value

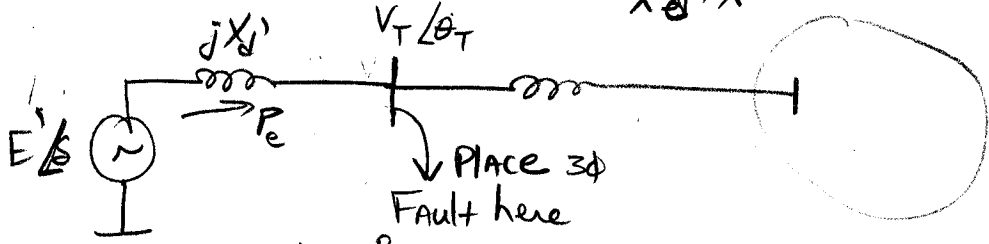
Why important?



$$P_e = \frac{E' V_e \sin \delta}{X_d' + X_e}$$



If E', V_e const, $P_{eMAX} = \frac{E' V_e}{X_d' + X_e}$



$$P_e = \frac{E' V_T \sin(\delta - \theta_T)}{X_d'} = 0$$

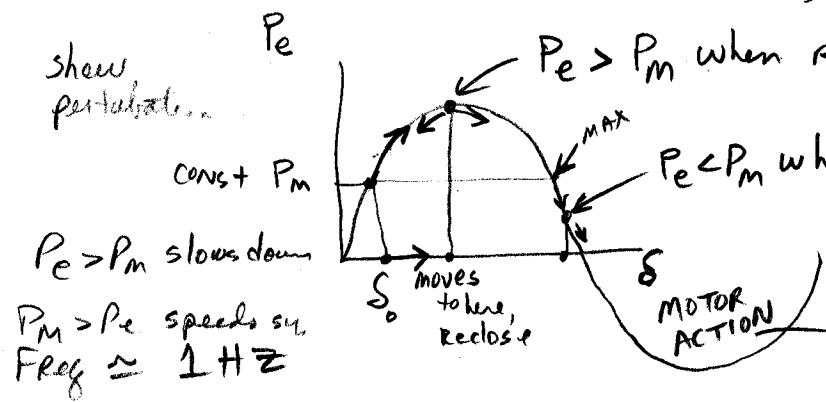
Note - very large current, but NO REAL power during fault

Swing Equation

$$\frac{d^2\delta}{dt^2} = \frac{\omega_s}{2H} [P_m - P_e] = \frac{\omega_s P_m}{2H}$$

Large pos number

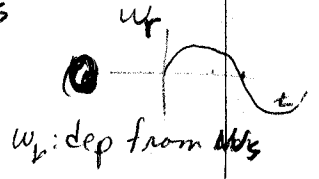
show perturbation



$\dot{\omega} < 0$ overshoots, but slows down
 $\dot{\omega} > 0$ keeps accelerating too late
 trip 108% Overspeed

MACHINE TRIPS OFF TO AVOID SERIOUS OVER CURRENTS

what is the MAX reclose δ for stability?



During Fault t

$$W = W_s + W_r$$

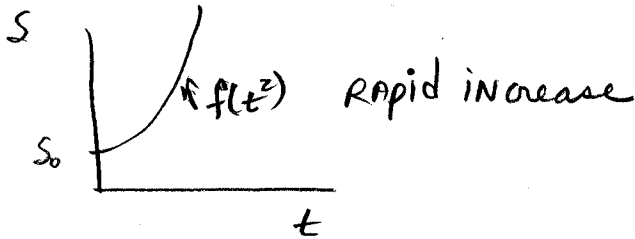
\downarrow
const

A3/2

55

$$W_r = \frac{dS}{dt} = \int_0^t \frac{d^2S}{dt^2} = \frac{W_s P_m t}{2H} \quad (\text{Above synch})$$

$$S = \int W_r dt = \frac{W_s P_m t^2}{4H} + S_0 \quad (\text{Above synch})$$



$$\frac{dW_r}{dt} = \frac{W_s P}{2H A}$$

$$W_r = \frac{W_s}{2H} \int P dt$$

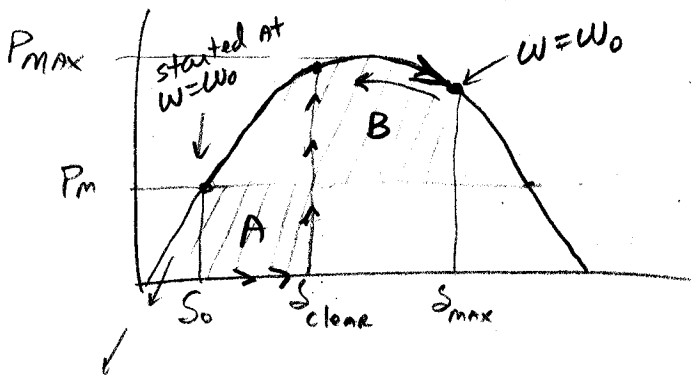
Starts & stops where $W_r = 0$

$$\int P_A dt = 0$$

$$dS = W_r dt$$

$$\int \frac{P_A dS}{W_r} =$$

EQUAL AREA criteria



$$\frac{dW_r}{dt} = \frac{W_s P_A}{2H} \rightarrow dW_r = \frac{W_s P_A}{2H} dt$$

start & finish out

$$\text{also, } dS = W_r dt \rightarrow dt = \frac{dS}{W_r} \Rightarrow dW_r = \frac{W_s P_A}{2H W_r} dS$$

$$W_r dW_r = \frac{W_s P_A}{2H} dS$$

Integrate both sides

$\int_{W_r=0}^{W_r=P} W_r dW_r = 0$ since W_r same integration limits

$$\therefore \int_{S_0}^{S_{max}} P_A dS = 0 \quad \text{Equal area criteria}$$

$$A = B$$

$$A = P_m (S_{clear} - S_0)$$

$$B = \int_{S_{clear}}^{S_{max}} (P_e - P_m) dS$$

But $P_e = P_{MAX} \sin S$

$$B = P_{MAX} \int_{S_{CLEAR}}^{S_{MAX}} \sin S ds - P_M (S_{MAX} - S_{CLEAR})$$

$$= P_{MAX} [\cos(S_{CLEAR}) - \cos(S_{MAX})] - P_M [S_{MAX} - S_{CLEAR}]$$

A = B

$$P_M S_{CLEAR} - P_M S_0 = P_{MAX} [\cos(S_{CLEAR}) - \cos(S_{MAX})] - P_M S_{MAX} + P_M S_{CLEAR}$$

$$\cos(S_{CLEAR}) = \frac{P_M}{P_{MAX}} [S_{MAX} - S_0] + \cos(S_{MAX})$$

WHAT IS S_{MAX} allowed? (Intersection of P_M)
 $= \pi - S_0$

Also, $P_M = P_{MAX} \sin S_0$

$$\cos(S_{CLEAR}) = \sin S_0 [\pi - S_0 - S_0] + \underbrace{\cos(\pi - S_0)}_{= -\cos S_0}$$

$$S_{CLEAR} = \cos^{-1} \left[(\pi - 2S_0) \sin S_0 - \cos S_0 \right] \quad \text{critical clearing}$$

How long?

$$S_{CLEAR} = \frac{\omega_s P_M t_{CLEAR}^2}{4H} + S_0$$

$$t_{CLEAR} = \sqrt{\frac{4H (S_{CLEAR} - S_0)}{\omega_s P_M}}$$

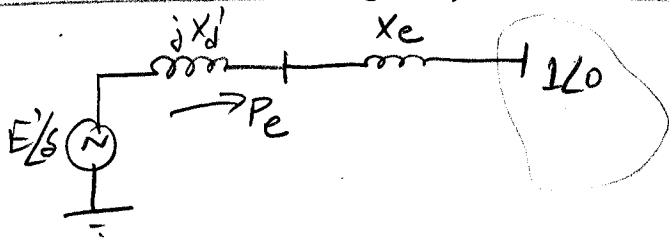
$\frac{RAD}{SEC}$ \rightarrow H, P_M, S_0 important
 $\frac{MVA}{MVA} \rightarrow$ \uparrow \uparrow \uparrow
 (PU) \uparrow \uparrow \uparrow
 RAD

critical clearing time

$$\leq 10 \text{ ms}$$

$$\approx .166 \text{ sec}$$

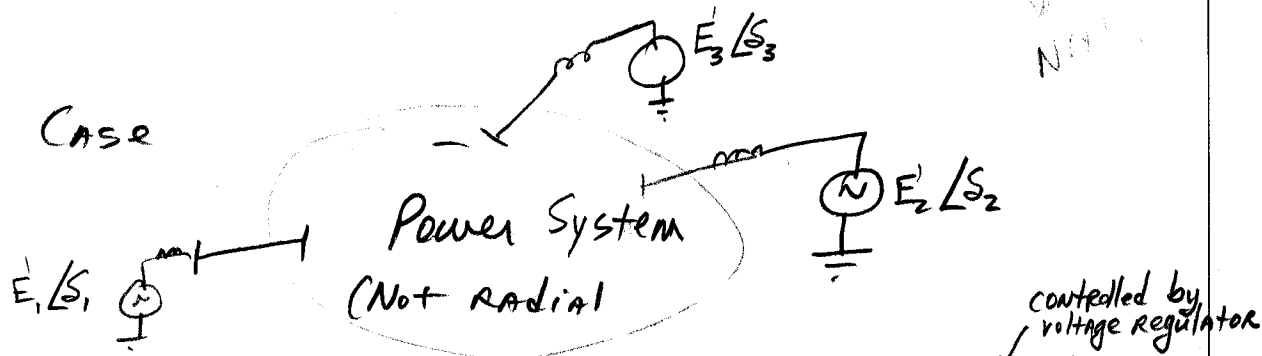
Solution of Swing Equation



$$\frac{d^2\delta}{dt^2} = \frac{W_s}{2H} [P_m - P_e] = \frac{W_s P_m}{2H} - \frac{W_s P_{max}}{2H} \sin\delta$$

Nonlinear D.E.

GEN CASE



$$\frac{2H^N}{W_s} \frac{d^2\delta_N}{dt^2} = P_m^N - P_e^N$$

Function of E, δ

either constant or controlled by steam/governor system

How to get P_e

If loads are represented as constant Z's to gnd, knowing E's, δ 's will yield all network voltages & currents \rightarrow powers (w/o loadflow)
 P_e

Constant Power Loads

Run loadflow with E, δ known. Solve for powers P_e

Constant Current Loads ?