

EE 394V ATQ5 Control. (Part 1)

Note Title

8/29/2007

Review

⊗ $\ddot{z} + a\dot{z} + bz = c\mu$

$x_1 = z, \quad x_2 = \dot{x}_1 = \dot{z} \rightarrow \dot{x}_1 = x_2$

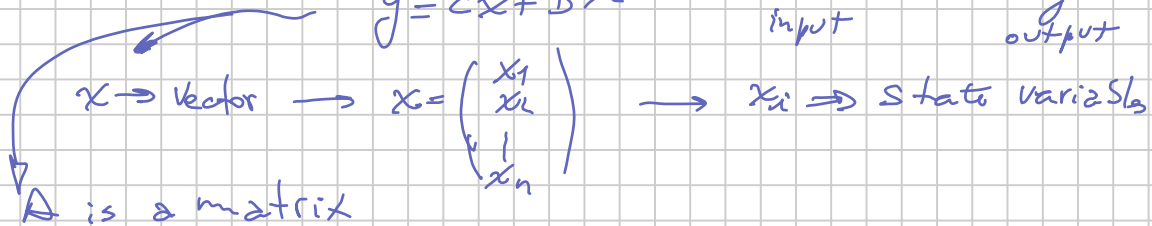
$\dot{x}_2 + a x_2 + b x_1 = c\mu \rightarrow \dot{x}_2 = -b x_1 - a x_2 + c\mu$

$\dot{x} = \begin{pmatrix} \dot{x}_1 \\ \dot{x}_2 \end{pmatrix} = \underbrace{\begin{pmatrix} 0 & 1 \\ -b & -a \end{pmatrix}}_A \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} + \underbrace{\begin{pmatrix} 0 \\ 1 \end{pmatrix}}_B \mu$

State space representation ^A of a linear ^B system:

⊗

$\dot{x} = Ax + B\mu$
 $y = Cx + D\mu$



For nonlinear systems: $\dot{x} = f(x, \mu)$

Equilibrium point(s) \rightarrow (only one or many connected in linear systems)

$\Leftrightarrow \dot{x}(x_q) = 0$

\rightarrow Recall that \dot{x} is like z velocity so equilibrium means $\dot{x} = 0$ (no velocity)

Other ways of representing linear systems is with a transfer

function \rightarrow

⊗

$G(s) = \frac{Y(s)}{U(s)}$

\rightarrow Impedance (Assume by now SISO system)

$Xs = Ax + Bu \rightarrow X(Is - A) = BU$

$Y = CX + DU$

\downarrow
 $X = (Is - A)^{-1} BU$

$Y = C(Is - A)^{-1} BU + DU$

$Y = (C(Is - A)^{-1} B + D)U$

Impedance $\Rightarrow G(s) = \frac{Y}{U} = C(sI - A)^{-1} B + D$

→ output

→ input

For $\dot{x} = Ax + Bu$, $x(t=0) = x_0$, the solution (trajectory)

$$x(t) = \phi(t, t_0) x_0 + \int_{t_0}^t \phi(t, \sigma) B(\sigma) u(\sigma) d\sigma$$

$$x(t) = \underbrace{\phi(t, t_0) x_0}_{\text{related with transient}} + \underbrace{\int_{t_0}^t \phi(t, \sigma) B(\sigma) u(\sigma) d\sigma}_{\text{related with steady state}}$$

related with transient

related with steady state

↓
driven by internal energy

↓
driven by input

related with states variables

For A non dependent on time

$$\phi(t, t_0) = e^{A(t-t_0)}$$

→ Matrix exponential

$$\hookrightarrow e^{At} = \sum_{i=0}^{\infty} \frac{A^i t^i}{i!}$$

In linear systems the solution for the state variables takes the form:

$$x_i(t) = \sum_{j=1}^n C_j e^{\lambda_j t} + X(u, t)$$

steady state

transient

$\lambda_j \rightarrow$ eigenvalues \rightarrow obtained from solving

$$\det(\lambda I - A) = 0$$

$$\leftarrow (A - \lambda I)x = 0$$

characteristic polynomial: $\mathcal{P}(\lambda) \hat{=} \det(\lambda I - A) = 0$

Example of eigenvalue use \rightarrow Stability $\rightarrow \text{Re}(\lambda) < 0 \rightarrow$ stable
 \hookrightarrow At least one $\text{Re}(\lambda) > 0$ unstable

So let's go back to power electronics as an application.
 Consider first the buck converter

Buck converter:

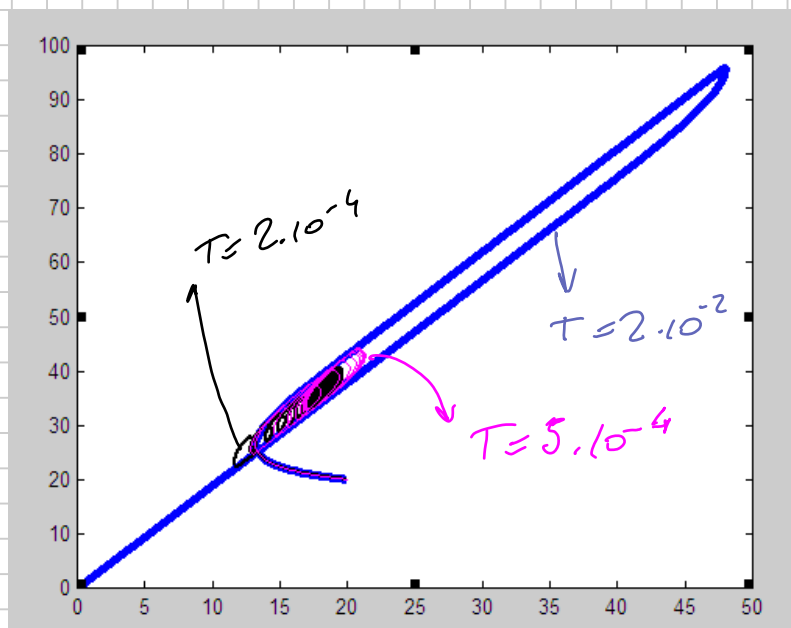
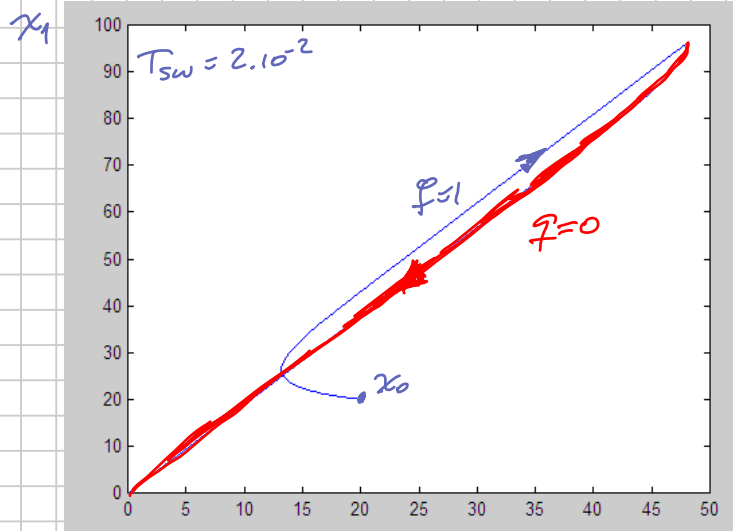
Switched system

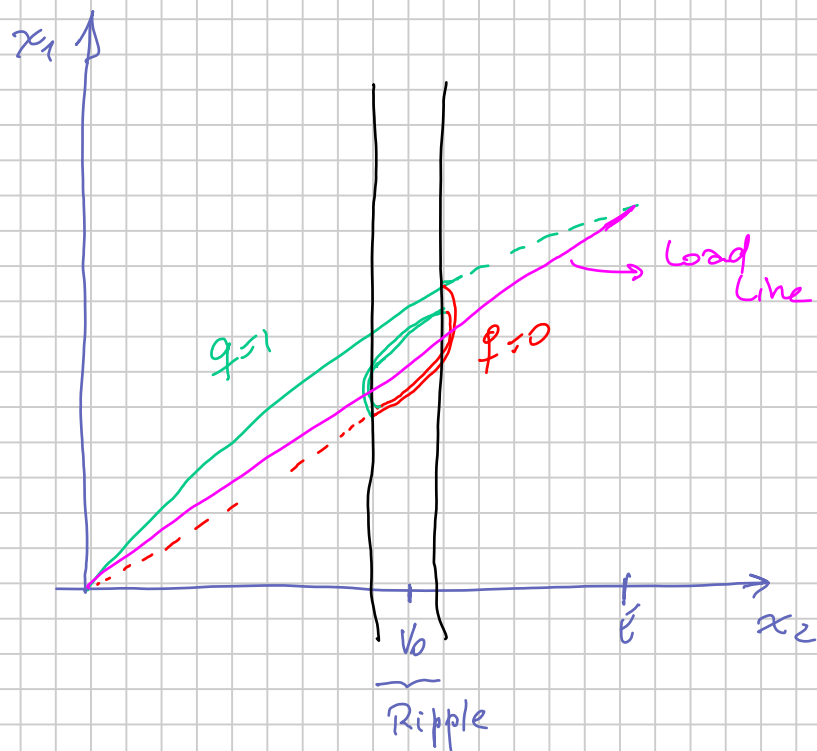
$$\begin{cases} L \dot{x}_1 = f(t) E - x_2 \\ C \dot{x}_2 = x_1 - \frac{x_2}{R} \end{cases} \quad x(0) = x_0$$

Equilibrium

For $f(t) = 0 \rightarrow x_2 = 0, x_1 = 0$

For $f(t) = 1 \rightarrow x_2 = E, x_1 = \frac{E}{R}$





Fast average

$$(1) \begin{cases} L \dot{x}_1 = dE - x_2 \\ C \dot{x}_2 = x_1 - \frac{x_2}{R} \end{cases}$$

$$\longrightarrow M \dot{x} = Ax + Bu$$

$$M = \begin{pmatrix} L & 0 \\ 0 & C \end{pmatrix}, \quad A = \begin{pmatrix} 0 & -1 \\ 1 & -1/R \end{pmatrix}$$

Based on control input $\rightarrow B = \begin{pmatrix} E \\ 0 \end{pmatrix}, \mu = d$

Based on power input $\rightarrow B = \begin{pmatrix} d \\ 0 \end{pmatrix}, \mu = E$

or

$$\begin{cases} \dot{x}_1 = \frac{1}{L} (dE - x_2) \\ \dot{x}_2 = \frac{1}{C} (x_1 - \frac{x_2}{R}) \end{cases} \longrightarrow \dot{x} = \begin{pmatrix} 0 & -\frac{1}{L} \\ \frac{1}{C} & -\frac{1}{RC} \end{pmatrix} x + \begin{pmatrix} E/L \\ 0 \end{pmatrix}$$

Equilibrium point:

$$x_{20} = dE$$

$$x_{10} = \frac{x_{20}}{R} = \frac{D_0 E}{R}$$

Since we want x_{20} constant (dc) then $d(t) = D_0$
 \downarrow
 Constant

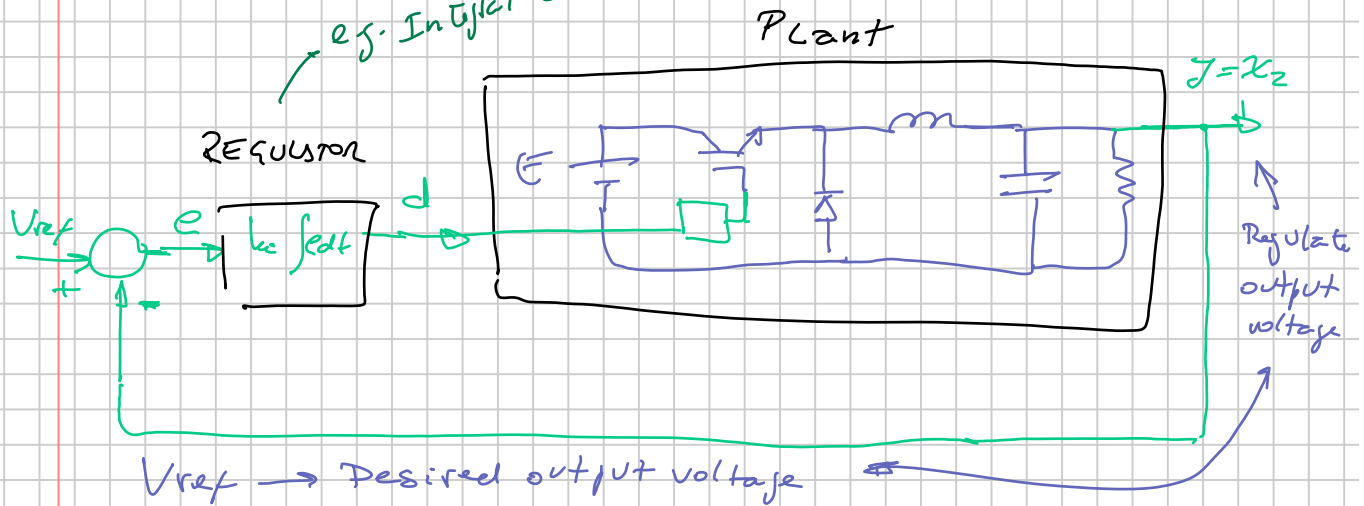
Regulation vs. Tracking

with respect to
an operation point

Follow some trajectory

Linear controller

eg. Integral controller



$$e = V_{ref} - x_2$$

$$d = k_i \int e dt = k_i \int (V_{ref} - x_2) dt$$

If $x_2 < V_{ref} \rightarrow d$ increases, $e > 0$

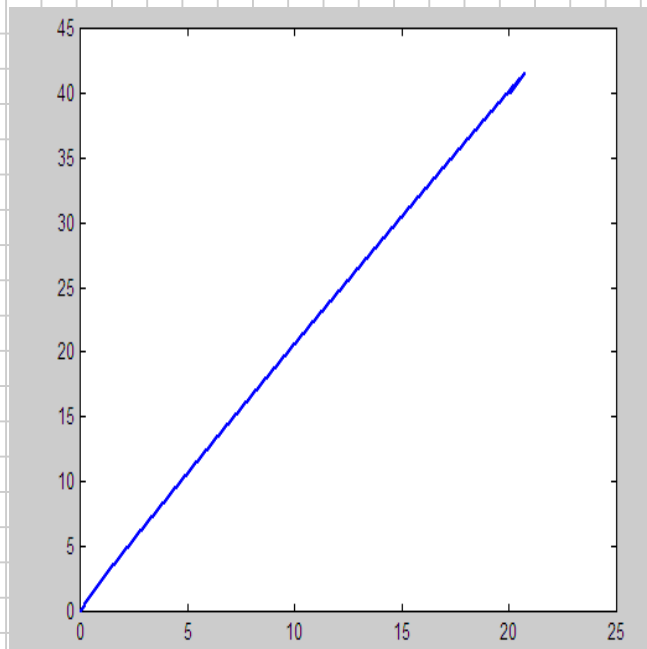
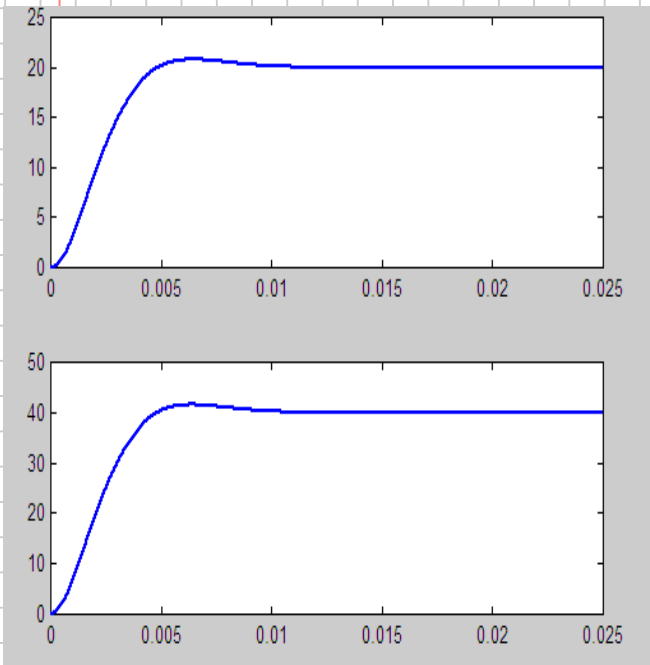
If $x_2 > V_{ref} \rightarrow d$ decreases, $e < 0$

If $x_2 = V_{ref} \rightarrow d$ constant, $e = 0$

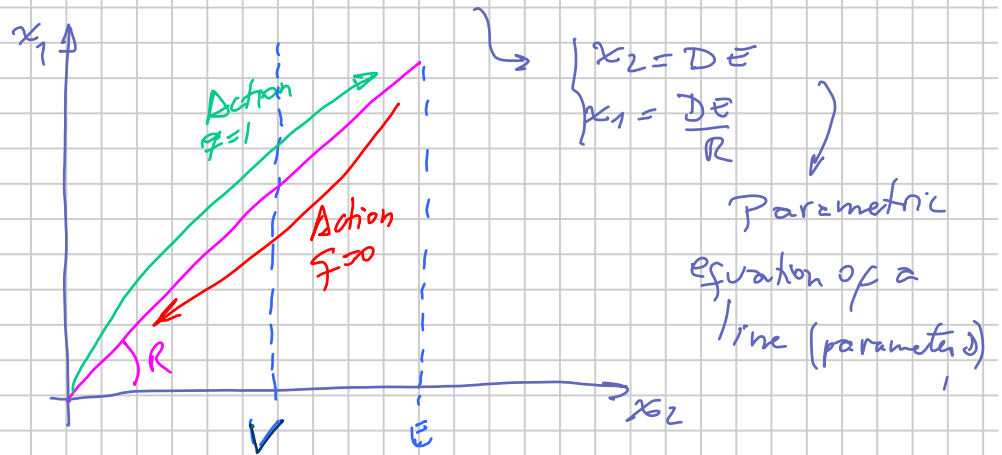
If $e = 0 \Rightarrow \dot{x}_2 = 0$
(because $\dot{e} = \dot{x}_2$)

Thus $e = 0$ and $d = \text{constant}$ at the equilibrium point

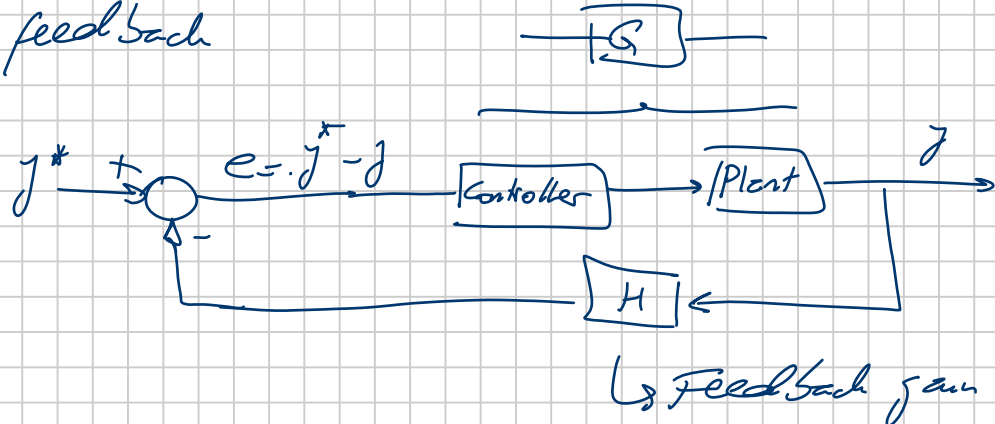
Same example, $k_i = 10$



From
$$\begin{cases} L\dot{x}_1 = DE - x_2 \\ C\dot{x}_2 = x_1 - \frac{x_2}{R} \end{cases} \Rightarrow x_1 = \frac{x_2}{R} \text{ (Line)}$$



More on feedback
Consider



Closed loop gain $\rightarrow T = \frac{Y}{Y^*} = \frac{G}{1+GH}$

Bandwidth \rightarrow Max frequency of a disturbance for which there is enough gain to compensate it

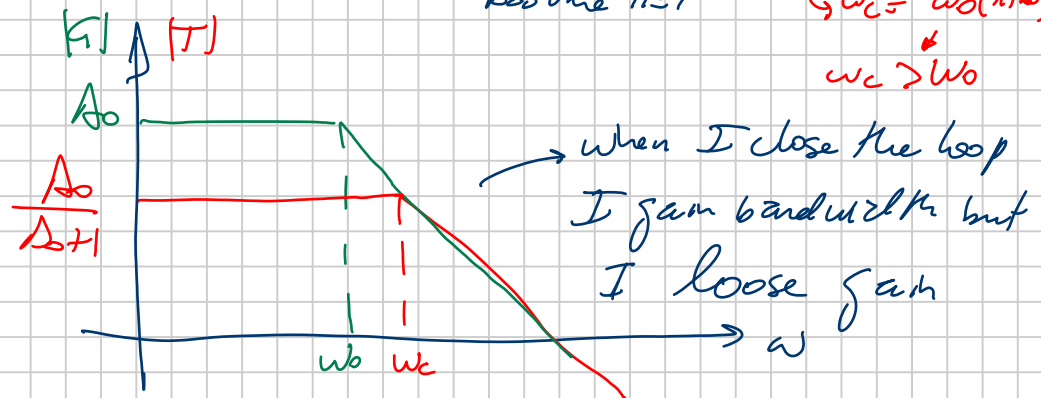
\hookrightarrow It is the frequency at which $T(s)$ dropped 3dB with respect to its low frequency value

Consider that open loop I have $\rightarrow G = \frac{A_0}{1+s/\omega_0}$

Closed loop $\rightarrow T = \frac{G}{1+GH} = \frac{A_0}{1+A_0+s/\omega_0}$

Assume $H=1$

$\hookrightarrow \omega_c = \omega_0(1+A_0)$
 $\omega_c > \omega_0$



Stability definitions (linear approach)

Crossover frequency $\omega_{co} \rightarrow$ 1st frequency at which $|G(s)H(s)| = 0dB$

$\omega_{180} \rightarrow \text{Arg}(G(s)H(s)) = 180^\circ$

$|G(j\omega_{180})H(j\omega_{180})| = 1 \rightarrow$ oscillatory

$|G(j\omega_{180})H(j\omega_{180})| > 1 \rightarrow$ unstable

$|G(j\omega_{180})H(j\omega_{180})| < 1 \rightarrow$ stable

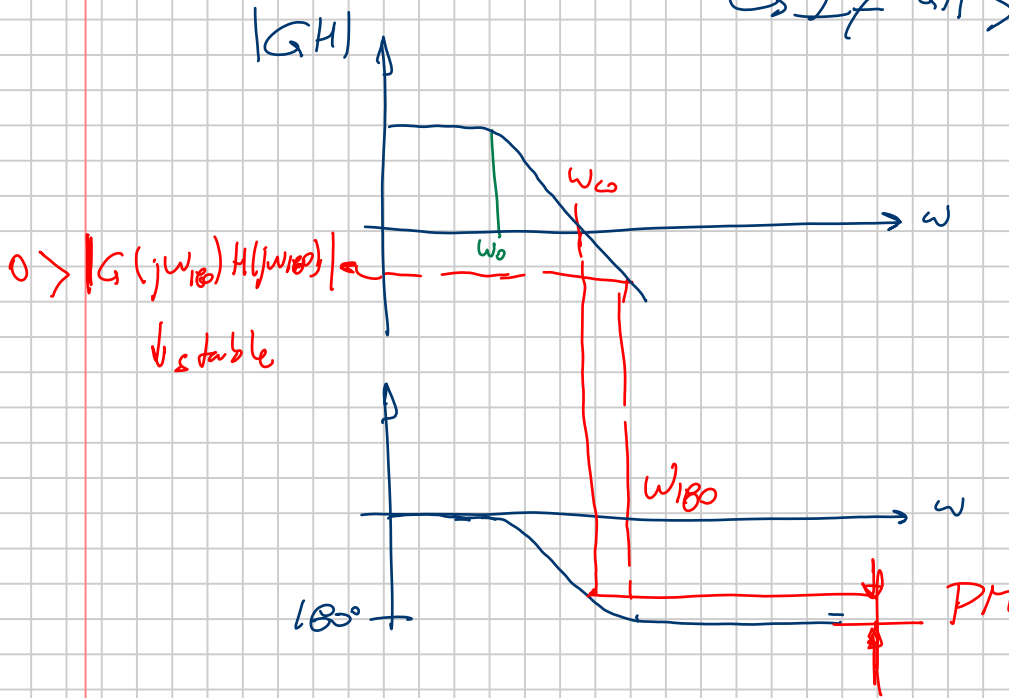
Phase margin $\rightarrow PM = \text{Arg} (G(j\omega_c) H(j\omega_c)) + 180^\circ$

\hookrightarrow If $PM > 0 \rightarrow$ stable

Gain margin \rightarrow Represents how much I can increment the system's gain without losing stability

$$GM = -|G(j\omega_{180}) H(j\omega_{180})|$$

\hookrightarrow If $GM > 0 \text{ dB} \rightarrow$ stable



Linear controllers

P control

$$e \rightarrow \boxed{G} \rightarrow \dot{y} = e \rightarrow \boxed{k_p} \rightarrow d \rightarrow \boxed{\text{Plant } A(s)} \rightarrow \dot{y}$$

$G \cdot \dot{y} = A(s) \cdot d$

$$d = k_p e = k_p (y^* - y)$$

Now, I close the loop with $H=1$

$$\frac{\dot{y}}{y^*} = \frac{G}{1+GH} = \frac{A(s)}{\frac{1}{k_p} + A(s)}$$

Notice that as $k_p \rightarrow \infty \Rightarrow \frac{\dot{y}}{y^*} = 1 \Rightarrow y = y^*$ \rightarrow My output equals my desired goal

So a high k_p yields a good tracking (hence regulation)

But \rightarrow If I have noise then
(little noise) \times (big k_p) = lots of noise

$d = k_p(y^* - y) \rightarrow$ large $k_p \Rightarrow$ yield $d > 1$
since d can only take values between 0 and 1 in reality
a large k_p is ineffective

\rightarrow large d s are bad for boost, buck-boost and other type of converters

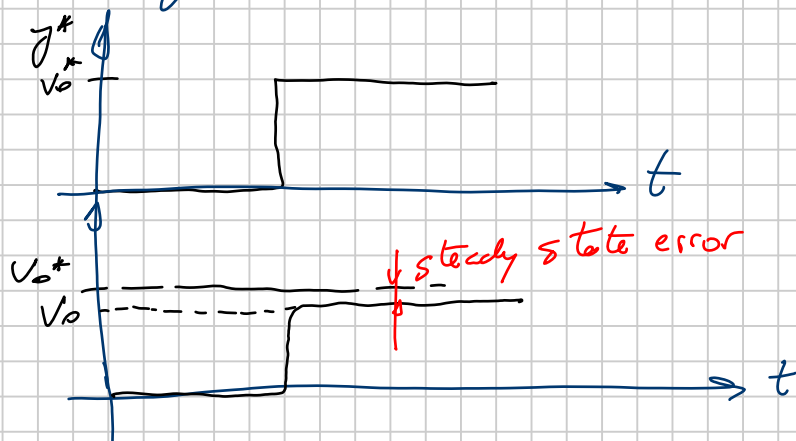
If $e = 0$ then $d = 0 \rightarrow$ So when I am meeting my control goal I have no control signal

Large k_p improves convergence to goal but worsens PM.

There are 2 important issues in proportional controls

$y = y^*$ only when k_p is ∞ , but ∞ in practice doesn't exist
when $e = 0$, $d = 0$

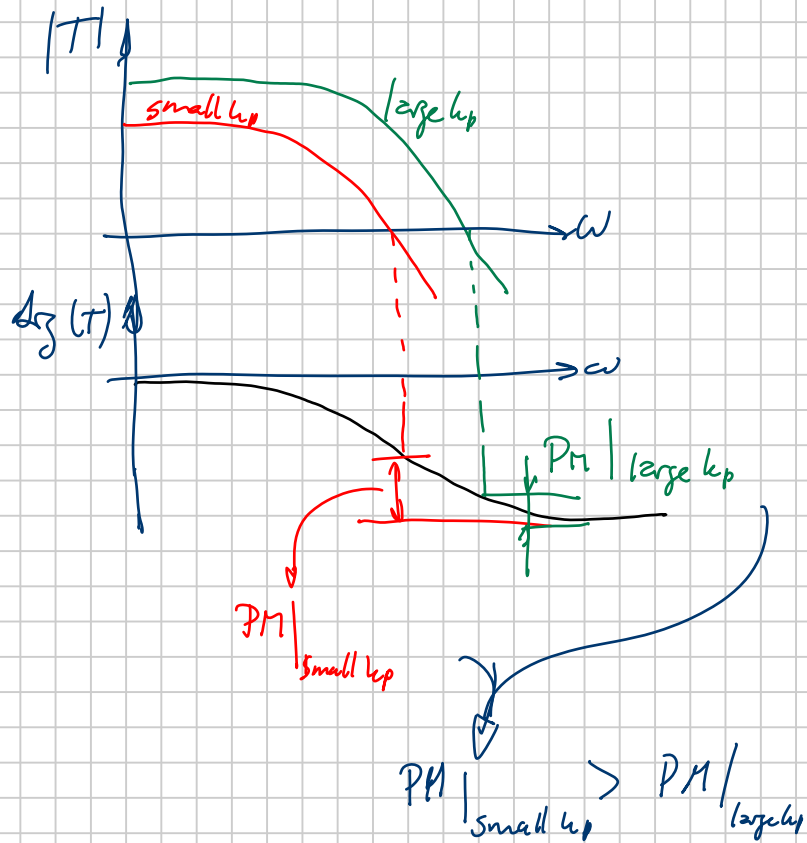
\rightarrow In proportional controllers I end up with a steady state error



Another problem of P-control is that large k_p s lead to lower PM

Ge.g. \rightarrow Assume that $A(s) = \frac{1}{(s+a)(s+b)}$

$G_H = \frac{k_p}{(s+a)(s+b)}$

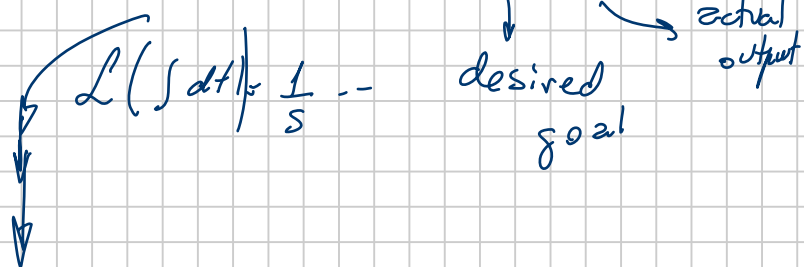


I-control

$e \rightarrow \left[\frac{1}{s} \right] \rightarrow \dot{y} = e \rightarrow \left[k_p \right] \rightarrow d \rightarrow \left[\frac{\text{Plant}}{A(s)} \right] \rightarrow y$

$\hookrightarrow \dot{y} = A(s)D$

$d = d_0 + k_i \int e dt = k_i \int (y^* - y) dt + d_0$



$$\frac{y}{y^*} = \frac{A(s) k_i/s}{1 + A(s) k_i/s} = \frac{A(s) k_i}{s + A(s) k_i}$$

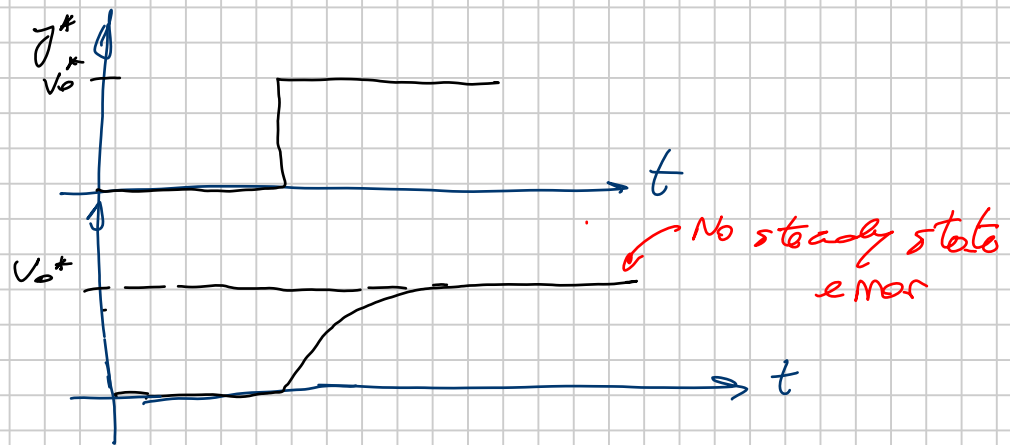
In the same way that $T \rightarrow \infty$ implies $\omega \rightarrow 0$ then
 $t \rightarrow \infty$ implies $s \rightarrow 0$
 \hookrightarrow steady state \hookrightarrow dc.

So with PI control at dc $\frac{y}{y^*} \Big|_{dc} = 1$

And as $t \rightarrow \infty$ (regardless of dc or ac signal)

$$y(t) \xrightarrow{t \rightarrow \infty} y^*(t) \text{ so } \lim_{t \rightarrow \infty} e = 0$$

Since $d = d_0 + k_i \int e dt \rightarrow d \neq 0$ when $e = 0$ even if $d_0 = 0$ because the integral is a "sum" so the result at $t \rightarrow \infty$ when $e = 0$ is not necessarily 0 because of all the contributions to the sum from $t = 0$ to $t = \infty$

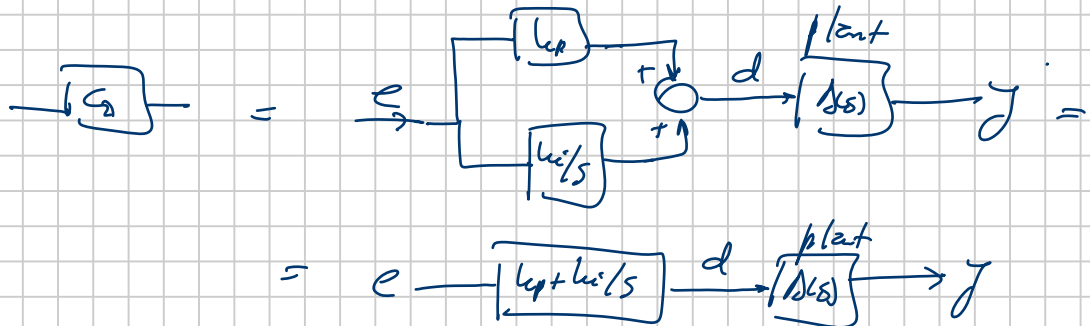


If $A(s) = \frac{A_0}{s+p} \rightarrow T = \frac{k_i A_0}{s^2 + sp + k_i A_0}$

$$d_{1,2} = \frac{-p \pm \sqrt{p^2 - 4k_i A_0}}{2}$$

- Hence,
- 1) If k_i is very large we can get complex solutions (in the frequency domain) which yield oscillatory behavior and overshoots in time domain
 - 2) Provided that $k_i > 0$, if $p > 0$ (o.l. stable) then the closed loop equilibrium point is also stable.

PI control



$$\frac{y}{y^*} = \frac{s(k_p \Delta(s)) + k_i \Delta(s)}{s(1 + k_p \Delta(s)) + k_i \Delta(s)}$$

So as $t \rightarrow \infty$ and $s \rightarrow 0$ then $\frac{y}{y^*} = 1 \rightarrow e = 0$ as $t \rightarrow \infty$

PI controllers can achieve faster convergence without having large k_i 's that may create overshoots or oscillations, but without ending up with steady state error

Boost converter

$$\begin{cases} L\dot{x}_1 = E - \frac{1}{\gamma} x_2 \\ C\dot{x}_2 = \frac{1}{\gamma} x_1 - \frac{x_2}{R} \end{cases}$$

For $g'(4) = 0$

$$\begin{cases} L\dot{x}_1 = E \\ C\dot{x}_2 = \frac{x_2}{R} \end{cases} \xrightarrow{\text{eq. point}} \begin{cases} 0 = E? \\ x_1 = ? \end{cases}$$

If $g'(t) = 0 \Rightarrow g(t) = 1$

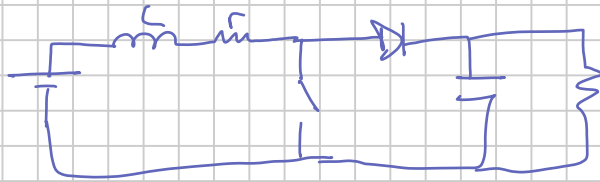


$L\dot{x}_1 = E \Rightarrow$ No equilibrium

$x_1(t) = \frac{E}{L}t + \frac{E}{L}x_1(0)$

Linear increase tending to infinity

Now assume that the inductor has some resistance r

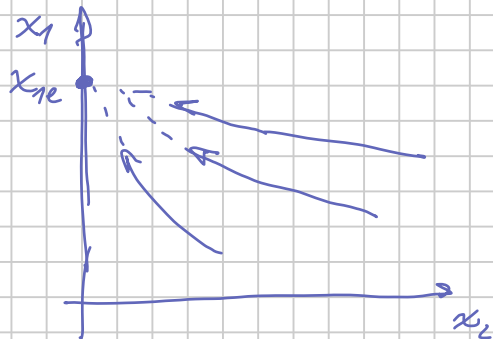


$$\begin{cases} L\dot{x}_1 = r x_1 - g'(t) x_2 + E \\ C\dot{x}_2 = g'(t) x_1 - \frac{x_2}{R} \end{cases}$$

For $g'(t) = 0 \rightarrow g(t) = 1$

$$\begin{cases} L\dot{x}_1 = r x_1 + E \rightarrow x_{1,eq} = E/r \\ C\dot{x}_2 = -\frac{x_2}{R} \rightarrow x_{2,eq} = 0 \end{cases}$$

$\hookrightarrow \dot{x}_2 < 0$
 $\dot{x}_1 > 0$



For $g'(t) = 1 \rightarrow g(t) = 0$

$$\begin{cases} L\dot{x}_1 = -r x_1 - x_2 + E \\ C\dot{x}_2 = x_1 - \frac{x_2}{R} \end{cases} \rightarrow x_{1,eq} = \frac{x_{2,eq}}{R}$$

$-\frac{r}{R} x_{2,eq} - x_{2,eq} + E = 0$

$\hookrightarrow x_{2,eq} = \frac{E}{1 + r/R}$

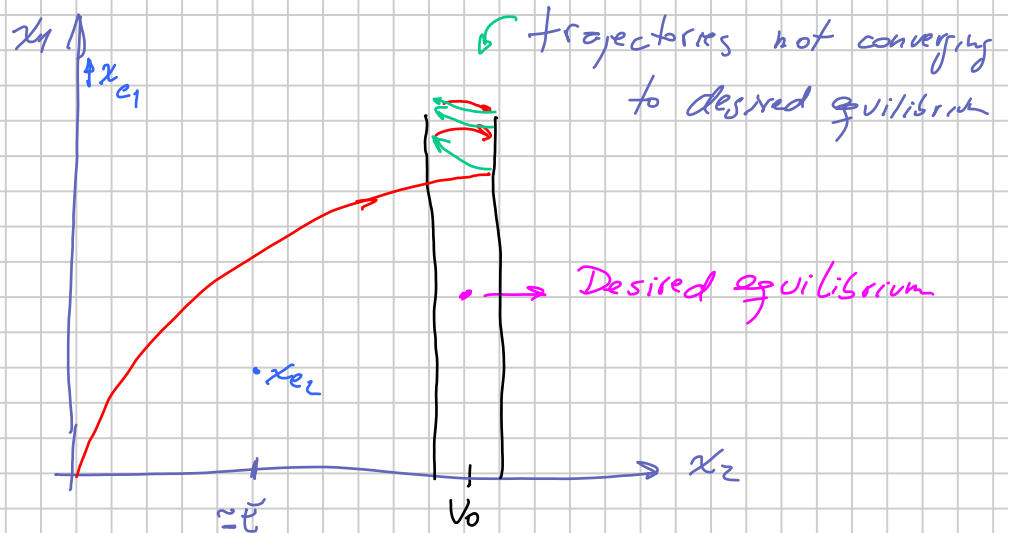
$x_{ref} = E$ because $r \ll R$



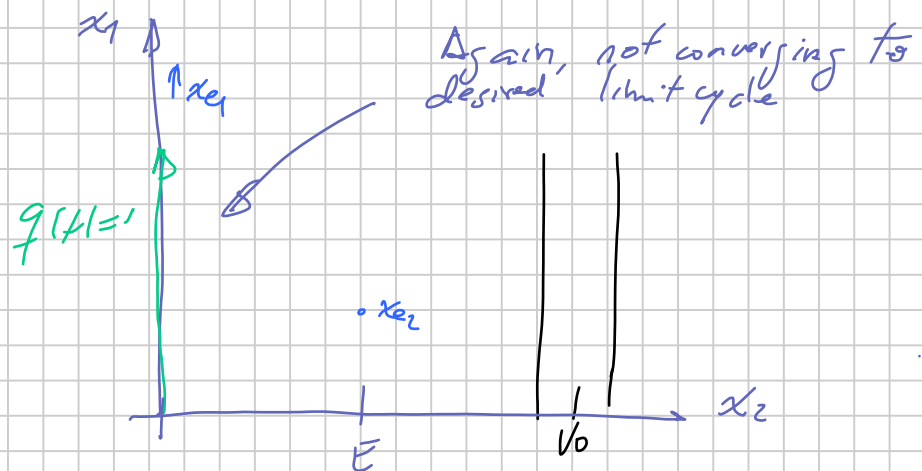
lets attempt \rightarrow geometric control to regulate output voltage

Consider $x(0) = 0$

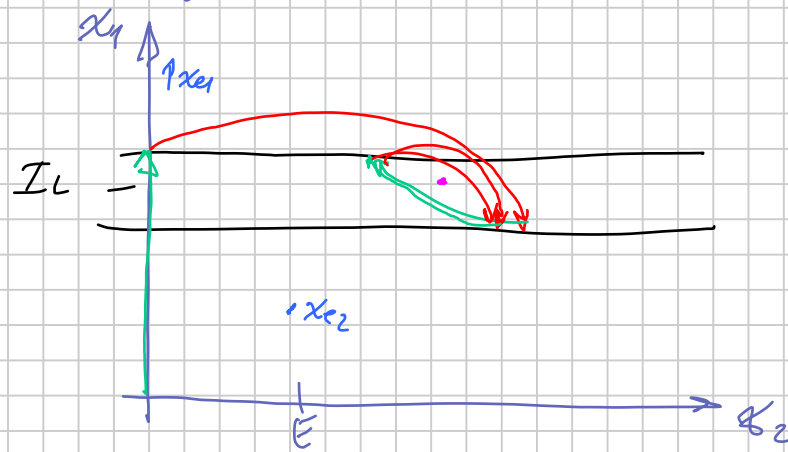
a) consider that the first control action is $g(t) = 0$



b) Now consider that the first control action is $g(t) = 1$



Now, lets attempt to regulate inductor current



Now, we achieved the desired goal

"Rules" for hysteresis control:

- 1) The switching surfaces (curves or lines) must separate the equilibrium points
- 2) The switches must act in opposition to the natural evolution toward an equilibrium point
- 3) The switch action must have a direct action on the regulated state variable
- 4) The desired operating point must be between switching surfaces
- 5) A dead band should be provided to avoid chattering

PI cont:

$$d = k_i \int e dt + k_p e$$

↳ Provides a similar action but faster than Integral term



average

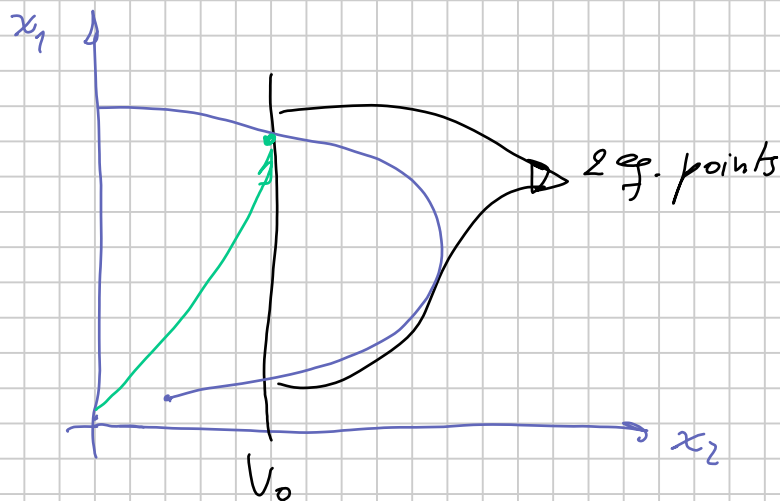
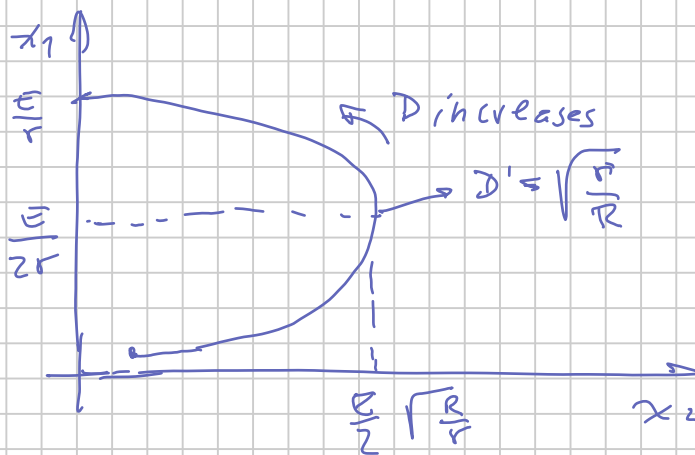
$$\begin{cases} L\dot{x}_1 = rx_1 - d'x_2 + E \\ C\dot{x}_2 = d'x_1 - \frac{x_2}{R} \end{cases}$$

Suppose $d' = D'$

$$x_1 = \frac{1}{r} (E - D'x_2)$$

$$x_1 = \frac{1}{D'} \frac{x_2}{R} \rightarrow D' = \frac{x_2}{x_1 R}$$

$$x_2 = \sqrt{-x_1^2 Rr + ERx_1}$$



PI controller for boost converter

$$\begin{cases} L\dot{x}_1 = rx_1 - d'x_2 + E \\ C\dot{x}_2 = d'x_1 - \frac{x_2}{R} \end{cases}$$

PID controller $\rightarrow d = k_i \int e dt + k_p e$

k_i and k_p have to be small (slow controller) or d should be limited to a max value of $\approx 0.05 - 0.9$
Otherwise the trajectory goes to the equilibrium point with high k_p

Next: A more challenging case \rightarrow constant power loads